

AD A 088279



# DEPARTMENT OF DEFENCE DEFENCE SCIENCE AND TECHNOLOGY ORGANISATION AERONAUTICAL RESEARCH LABORATORIES

MELBOURNE, VICTORIA

STRUCTURES NOTE 454

A ONE-PASS METHOD FOR COUNTING RANGE MEAN PAIR CYCLES FOR FATIGUE ANALYSIS.

by

R. C. FRASER

Approved for Public Release.



-

C COMMONWEALTH OF AUSTRALIA 1979

COPY No ?

DOME T

80 8 27

033

# DEPARTMENT OF DEFENCE DEFENCE SCIENCE AND TECHNOLOGY ORGANISATION AERONAUTICAL RESEARCH LABORATORIES

#### STRUCTURES NOTE 454

# A ONE-PASS METHOD FOR COUNTING RANGE MEAN PAIR CYCLES FOR FATIGUE ANALYSIS

by

R. C. FRASER

#### SUMMARY

A one pass method for counting Range Mean Pair cycles is described. The Range Mean Pair Table which is used to represent the data generated by the method is considered with reference to its use in fatigue analysis.

# DOCUMENT CONTROL DATA SHEET

1. Document Nu (a) AR Numb AR-001-7- (b) Document Structures (c) Report Nu ARL-Struc	er: 46 Series and N Note 454	lumber:		<ul> <li>2. Security Classification</li> <li>(a) Complete document:</li></ul>											
3. Title: A ONE FOR F	PASS MET ATIGUE A			COL	JNT	ING	RA	NGE	ME	AN P	AIR	CYC	LES		
4. Personal Author Fraser, R. C.	or(s):				5.	. Doc	ume e, 19		ate:						
					6.	Тур	e of	Repo	rt an	d Per	iod C	over	ed:		
7. Corporate Aut Aeronautical F	` '	·	8. Reference Numbers (a) Task:												
9. Cost Code: 27 7030		-			(b) Sponsoring Agency:										
10. Imprint: Aeronautical F Melbourne	Research Lab	oratorie	!s,		11.	. Cor (Tit	-		_	n(s) age(s)	):	_			
12. Release Limita Approved for	,		ent)												
12-0. Overseas:	N.O.	P.R.	1	Α		В		С		D		E			
13. Announcemen No limitations		(of the	info	rmatic	on o	n this	pag	e):	<b>L</b> -	<b>.</b>	L				
14. Descriptors: Fatigue (mater Loads (forces) Cycles	15. Cosati Codes: ting 2011 e (extremes) 1201 s (data) 1113														
16. , A one pass Mean Pair Tabl with reference to		d to rep	g Ra reser	it the c	1ean										
nun rejerence t	o use in ja	urgue a	autys	12.							· (, , , , ,		For		
												-/-	il		

A special

# **CONTENTS**

1. INTRODUCTION	1
2. CYCLE DEFINITION	2
3. END EFFECTS	3
4. RANGE MEAN PAIR TABLE	4
5. RANGE MEAN PAIR TABLE USE	6
6. CONCLUSION	7
REFERENCES	
FIGURES	
APPENDIX	
DISTRIBUTION	

#### 1. INTRODUCTION

The interpretation of loading environment remains fundamental to many fields of fatigue investigation:

- (1) life estimation;
- (2) fatigue test load selection;
- (3) comparison of load spectra and damage estimates between aircraft, mission type etc.;
- (4) load spectra prediction for future aircraft design;
- (5) sequence analysis;
- (6) crack growth analysis.

Of all the cycle counting methods that exist for this purpose<sup>1-5</sup> the Rainflow and Range Mean Pair methods are deemed the most generally useful from a theoretical point of view because both identify load cycles in terms of the stable cyclic stress-strain behaviour of the material concerned (i.e. turning points are paired that define closed hysteresis loops).<sup>5-9</sup> However, although simpler by definition, the multipass characteristic of existing range mean pair methods has meant that they are less efficient than the one pass rainflow method for use on other than short load records.

The present paper describes a one pass method for counting range mean pair cycles that can be applied to complex load histories of unspecified lengths. The storage of range mean pair data is also discussed with particular attention to the benefits afforded by recording such information in a table. It should be noted that where load is referred to in this paper, strain, normal load factor, stress, bending moment etc., are equally applicable.

#### 2. CYCLE DEFINITION

The basic method for the extraction of range mean pairs from a given load history is given in Reference 2 and is summarized below:

The method is to select and remove from a time ordered list of load maxima and minima (turning points), the adjacent pair having the smallest absolute difference. This is repeated until all possible pairs are removed. Each pair is then considered to constitute the peak and trough of one load cycle for which a mean and alternating load can be determined.

Though extremely simple, this procedure has an obvious limitation: it obtains only one range mean pair for each pass through a given record and thus cannot be efficiently applied to long complex histories in this form.

However, an immediate start to reducing the number of passes required is made when it is seen that this minimum difference definition identifies cycles which constitute perturbations of other larger cycles (i.e. turning points which relate to closed stress-strain hysteresis loops) and that a test based on this may be used to detect more than one range mean pair per pass. From Figure 1 it can be seen that a perturbation test may be expressed so:

for a sequence of four turning points (TP's) denoted TP(k = 3), TP(k = 2), TP(k = 1), TP(k) if

$$| TP(k-3) - TP(k-2) | \ge | TP(k-1) - TP(k-2) | \le | TP(k) - TP(k-1) |$$

$$= equation (A)$$

the cycle TP(k-2), TP(k-1) constitutes a range mean pair. This will hitherto be referred to as the 'four point test'.

By advancing through the load history and considering four turning points at a time, using (A), the number of range mean pairs obtained per pass is increased although several passes are still required to process the entire load history. The refinement necessary to obtain complete processing in a single pass is realized when (A) is used repetitively as follows:

As each turning point is passed it is loaded into a turning point stack and equation (A) used to test if it identifies the previous two turning points in the stack as a range mean pair.

If a range mean pair is not detected the next turning point in the load history is loaded into the stack and the process repeated until a range mean pair is found. When this occurs the range mean pair turning points are removed from the stack, the gap closed and equation (A) used again to detect as many range mean pairs as possible e.g. if the turning point TP(k) identifies the turning points [TP(k-1), TP(k-2)] as a range mean pair it may similarly detect the turning points [TP(k-3), TP(k-4)] as a range mean pair and so on. The sequence of points for which this repetitive pairing occurs is shown in Figure 2.

In this way cycle counting proceeds through the load history with the turning point stack being progressively loaded and emptied.

In the preceeding it has been shown that a four point test can be used to process a load history in a single pass. However, it can be demonstrated that the single pass characteristic itself is conducive to a further improvement in the actual test for a range mean pair.

Consider Figure 3 where the second sequence of Figure 1 has been reproduced. The four point test (equation A) would pair TP(k)1), TP(k)2) as before. Now suppose that the same 3) is in a different position such as in Figure 3b. In this situation exists except that the  $\Gamma P(\lambda)$ instance the one pass four point procedure would not reach TP(k) with the given sequence undisturbed since it would have removed the pair TP(k)3), TP(k = 2) when it reached 1). Thus the turning point TP(k) 3) can only lie where it is depicted in Figure 3a (i.e. below the load values of 1P(k)2) and TP(k)1)) if it is to remain in the history unpaired when the four point one pass method reaches TP(k). Hence the use of the fourth point, TP(k)is unnecessary in this situation and only the right hand portion of equation (A) need be used as the range pair test (hereafter called the three point test). The same argument applies to the mirror image of Figure 3 if "below" is replaced by "above" so that the three point test suffices for all cases. The decision to use either the four point or three point test in the one pass method is considered below.

#### 3. END EFFECTS

The end effect problem is basically due to the fact that every practical load history is of a finite length and thus there must exist in every load history turning points which cannot be identified as perturbations of larger cycles simply because the turning points of those larger cycles do not occur in the given record. Hence, every range mean pair method must leave at the end of processing some unpaired 'residual' turning points.

Consider Figure 6a where load histories (a) to (h) are depicted. Because the prior and subsequent load sequences for each history are unknown no range mean pair can be found in any of them (i.e. no corresponding closed stress strain hysteresis loop can be firmly identified without more information at the ends of the given sequences) and thus the conservative strategy of pairing maximum peak to minimum trough is usually adopted.

After the last turning point of the load history has been loaded into the TP stack and either the three or four point test used to check if it defines any range mean pairs, it and possibly other TP's representing those discussed above, will remain unpaired in the TP stack. (It should be noted that the number of TP's involved is usually very small, often only two or three, and that sequences (a) to (d) represent the residuals possible after a three point test has been used while for the four point test, (c) to (h) are also possible.)

When a four point test is used in the one pass procedure the turning point stack is emptied using the minimum trough to maximum peak method as already outlined, however when a three point test is in use, it is possible to unload the residual TP's in the stack without changing to a different pairing process. If one considers sequences (a) to (c) of Figure 6a again, it can be seen that pairing of the turning points in the stack at the end of the load history can be accomplished by loading a large 'dummy' turning point into the end of stack and using the three point test as before to pair right to left as shown in Figure 2. When the last TP is a peak the 'dummy' TP is a large—ve number and vice versa for a trough (e.g.—or—10<sup>30</sup>). This may necessitate further further adjustments as described below.

The advantages in using the three point test for both the main processing and the end effect correction are that the computer program written to implement the method is short and simple and the execution time is similarly short even on long load histories. The disadvantage is that

in some instances the pairing of the end effect sequence is unconservative i.e. minimum trough to maximum peak pairing does not occur.

When a load history contains an odd number of turning points one TP will obviously remain after pairing. When the four point test is used this TP will be the peak or trough closest to the mean of the 'residual' sequence, (e.g. one of the turning points at either end of sequences (e) to (h) in Figure 6a) and when the three point test is used it will be the largest peak or smallest trough in the load history. In the former instance the damage contribution is slight and can usually be ignored, however in the latter case the damage contribution may be significant enough to warrant adding a mean load TP to the TP stack to ensure its pairing (this is sometimes called closing the sequence). When this nominal TP is used it is added to the stack before the dummy to obtain conservative pairing.

For a data sequence consisting of more than one block (flight) two alternatives exist for the application of the end effect correction. It may be used at the end of each block or at the end of the entire sequence. The choice of either alternative is basically a philosophical one, and may depend on many factors such as the accuracy of the data record in representing local loading conditions e.g. for a sequence of many flights of data over which there was little change in structural condition (no crack initiation or crack growth etc.) the latter alternative may be chosen. When the opposite is true it may be considered that applying the end effect correction at the end of each block results in some consistency in the results (i.e. turning points are paired which occur under similar conditions). The treatment of the 'odd' number turning point as discussed in the previous paragraph is also relevant here as is obvious that applying the end effect correction at the end of the entire sequence of blocks will result in only one possible 'odd' number turning point.

One other end effect requires some consideration. Should the first and last points in the record be considered as turning points? e.g. if Figure 7 represents an in-service load sequence it may be argued that points A and B constitute turning points though the influence of points A and B on the pairing is small in all cases except where the data record is very short. One convenient method of 'closing' a sequence uses point B as follows: if the turning point stack contains an odd number of turning points after the last true turning point has been loaded and used to detect as many range mean pairs as possible, then point B is considered a turning point and is loaded into the stack and used to test for range mean pairs. The pairing of the residual history proceeds as before for the specific test used. When the reverse is true and the stack contains an even number of turning points after the last true turning point has been considered then point B is not used and end correction proceeds.

Now consider the pairs obtained when the three point one pass procedure is used to cycle count each of the turning point sequences shown in Figure 6a using the 'nominal' and 'dummy' TP's as relevant. The results are shown in Figure 6b and for all sequences baring c, g, f maxpeak to min-trough type pairing occurs. The influence of the less conservative pairing demonstrated in sequences c, g, f on fatigue damage estimates is small for all but short load records. In the latter case a four point test is substituted for a three point test and a max-peak to min-trough pairing method used to pair off the g, e, f, h type sequences that will remain when all range mean pairs have been removed.

The complete one pass counting method obtained by correcting the basic procedure for end effects as above is shown schematically in Figure 8.

## 4. RANGE MEAN PAIR TABLE

Because of the large amount of RMP data that can be generated from long data records a means of recording such data efficiently is desirable. The range mean pair table fulfils this requirement and also provides a form which, as is shown in the next section, proves useful in many areas of fatigue analysis.

The table is simply a half array with axes of peak and trough load obtained by grouping the range mean pairs obtained from the load history into a number of cells.

Suppose that the maximum possible load existing in a given record will not exceed the value  $L_{\text{max}}$  and the minimum possible load will not be less than  $L_{\text{min}}$ . Then dividing this load range into n levels to give the level size LS, provides a basis for grouping the range mean pairs. Consider Figure 9 where the range mean pair of load x1 to load x2 is shown to be represented

on the basis of levels by the range mean pair of level (i+1) to level (i+5). Hence the cell in the range mean pair table corresponding to this range mean pair would record a count of one. At the end of processing of a load history all range mean pairs whose trough and peak were similarly in levels (i+1) and (i+5) respectively would be represented in the table as a corresponding count in the same cell (Fig. 10). Similarly all other range mean pairs generated by the counting method would be grouped into their respective cells in the range mean pair table.

When the information stored in the range mean pair table is required, the load data is calculated using the minimum load  $(L_{\min})$  and level size (LS) values e.g. the counts shown in Figure 10 represent range mean pairs from a trough of load  $L_{\min} + (i + 1 - 0.5)$ LS to a peak of load  $L_{\min} + (i + 5 - 0.5)$ LS. The mean and alternating loads can then be calculated from these values accordingly. It should be noted that:

- (i) The leading diagonal of the range mean pair table represents "degenerate" range mean pairs i.e. range mean pairs for which both the peak and trough lie within the one level. As the alternating load for these grouped range mean pairs is zero when determined by assigning load values to their peaks and troughs as above, they are not usually used in a fatigue damage calculation based on the range mean pair table (the S-N data used will determine if the range mean pair data contained in this diagonal should be included in the damage calculation in which case a conservative estimate of alternating load such as LS/4 could be used.)
- (ii) Diagonals parallel to the leading diagonal (down left to right) represent range mean pairs with the same alternating load.
- (iii) Conversely, diagonals in the opposite sense (up left to right) represent range mean pairs with the same mean load value, (Fig. 11).
- (iv) The range mean pair table shown in Figure 10 as a half array can also be configured as a vector to save computer storage space.
- (v) The number of levels into which the load range is divided determines the accuracy of the table in recording the range mean pairs discussed below.

In Figure 10 range mean pairs with troughs in level i + 1 and peaks in level i + 5 are shown recorded in the range mean pair table by the respective number of counts K. These range mean pairs are assumed to be distributed within the given levels such that their mean value in load terms can be taken to be the mean value of those levels. Thus the smaller the level size used (i.e. the larger the number of levels) the smaller the error inherent in this assumption. A typical example of the effect of the number of levels chosen for the table on its accuracy is illustrated in Figure 12 where fatigue damage estimated for a structural component has been calculated from the individual range mean pairs of an in service record and compared with that obtained from range mean pair tables of the same data. The 'zig-zagging' effect within the envelope shown in Figure 12 is a result of the range mean pairs suddenly crossing level boundaries as the number of levels within the tables is changed. Figure 12 also indicates the rapid convergence of damage estimates obtained from the tables to the correct value as the number of levels is increased. Experience has shown thirty or more levels to be preferable for range mean pair table damage estimates though sufficient accuracy is often obtained with as few as ten levels. The table's accuracy can be checked by comparing damage calculated at processing time with that obtained from the completed table.

#### 5. RANGE MEAN PAIR TABLE USE

The range mean pair table is used primarily for fatigue life estimation although it is useful in some of the other areas of fatigue interest given in the introduction.

Fatigue damage estimates can be obtained from the data contained in the table by calculating the damage attributable to each cell on the basis of its mean and alternating load and on the counts recorded therein, (degenerate diagonal cells are ignored) and summing in accordance with Miner's rule.

The range mean pair table also facilitates damage density calculations because of the way in which it presents ordered sets of mean and alternating load, (Fig. 11).

Fatigue meter counts of normal load factor form the basis of many in-service fatigue damage estimates. These counts can be simulated from range mean pair tables of vertical acceleration or related parameters. For a fatigue meter of x thresholds (where x is typically 8) the counts

recorded for each threshold can be found by summing all range mean pair counts within the area of the table bounded by those levels which encompass the corresponding 'cocking' and 'firing' levels, denoted respectively  $L_c$  and  $L_f$ . This is demonstrated in Figure 13 where the smallest range mean pairs capable of registering a count for the two types of thresholds ( $L_f > L_c$  and  $L_f < L_c$  respectively) are shown. Thus for either threshold type a range mean pair having a peak in a higher level and a trough in a lower level than the minimum required would also register a count for that threshold. Hence the total number of counts registered for the given fatigue meter threshold is the sum of all such range mean pairs in the table, i.e. the sum of all range mean pairs in the table bounded by the respective 'cocking' and 'firing' levels.

For a fatigue meter that 'fires' all thresholds at the same value (typically 1 g) the summation can be performed cumulatively. This is illustrated in Figure 14 for positive 'cocking' values. The same procedure is used to sum vertically for negative values.

Where the objective is not to simulate the performance of a particular fatigue meter but to provide data for spectra a slight modification is utilized. From Figure 15a summing proceeds cumulatively using every level in the range mean pair table (in effect representing a fatigue meter of n thresholds and variable 'cocking' and 'firing' values). This produces counts for spectra as shown in Figure 15b. Spectra for parameters other than normal load factor are produced in the same way as above from their respective range mean pair tables.

Two examples demonstrating the application of the one pass range mean pair method are given in the Appendix.

#### 6. CONCLUSION

A method for counting range mean pair cycles has been described that can be used to process a load history of any unknown length in a single pass. The obvious benefits of this method lie in its simple implementation, speed and application to unconditioned data, (i.e. no adjustment of a load history such as setting maximum load first etc. is required).

The range mean pair table which records data two dimensionally has also been discussed with particular attention to the manner in which is can be used to enhance the capabilities of the one pass method to process and store very large amounts of data.

#### REFERENCES

- 1. Schijve, J.—"The analysis of random load-time histories with relation to fatigue tests and life calculations". ICAF-AGARD Symposium, Paris, May 1961.
- 2. Dabell, B. J., and Watson, P.—"Cycle counting fatigue damage". Statistical aspects of fatigue testing Symposium, Warwick University, Feb. 1975.
- Endo, T., Kobayashi, K., Mitunaga, K., and Sugimua, N.—"Numerical comparison of the cycle count methods for fatigue damage evaluation, and plastic-strain damping energy of metals under random loading". 1975 Joint JSME-ASME Appl. Mech. Western Conf., 75– AM JSME A-17. (ICAF Doc. 825.)
- 4. de Jonge, J. B.—"The monitoring of fatigue loads". ICAS paper No. 70-31, 1970.
- 5. van Dijk, G. M.—"Statistical load data processing". National Aerospace Laboratory NLR The Netherlands, April 1971.
- Engineering Sciences Data Unit—"Fatigue life estimation under variable an plitude loading". ESDU Fatigue Sub Series, Item 77004.
- Fritz, J. T. D.—"An approach towards a study to determine the most realistic counting method in monitoring aircraft component fatigue life". CSIR Report ME 1384, April 1975
   S. Africa.
- 8. Tischler, V. A.—"A computer program for counting load spectrum cycles based on the range pair cycle counting method". Tech. Memo FBR 72-4, Nov. 1972. Air Force Flight Dynamics Lab., Ohio.
- 9. Sewell, R.—"An investigation of flight loads counting methods and effects on estimated fatigue life". NAE 1412-ST 431, Oct. 1970.
- 10. Ford, D., and Patterson, A. K.—"A range mean pair counter for monitoring fatigue". ARL Tech. Memo 195, Jan. 1971.

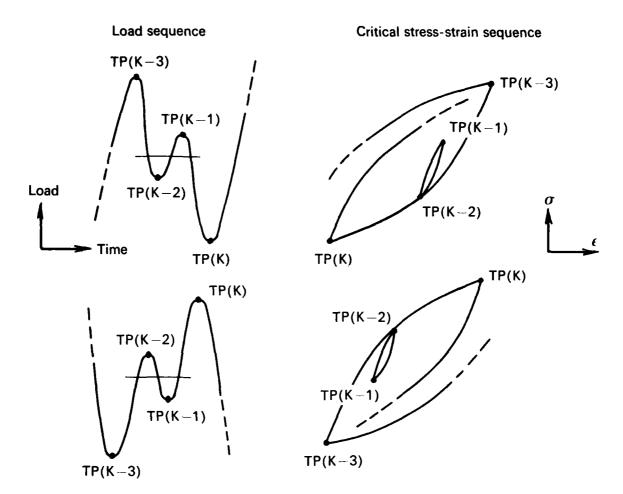


FIG. 1: THE PERTURBATION DEFINITION OF THE RANGE MEAN PAIR AND ITS CORRESPONDENCE TO STABLE CYCLIC STRESS-STRAIN HYSTERESIS LOOPS

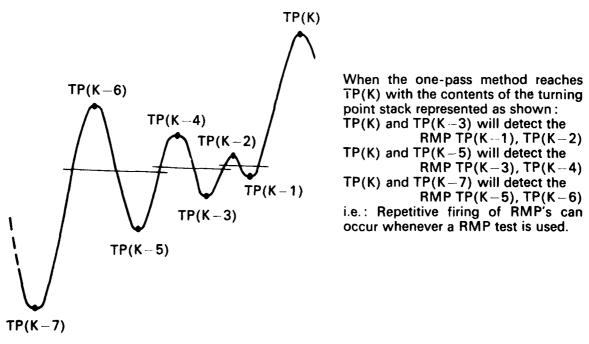


FIG. 2: REPETITIVE PAIRING OF RANGE MEAN PAIR CYCLES BY A ONE-PASS METHOD

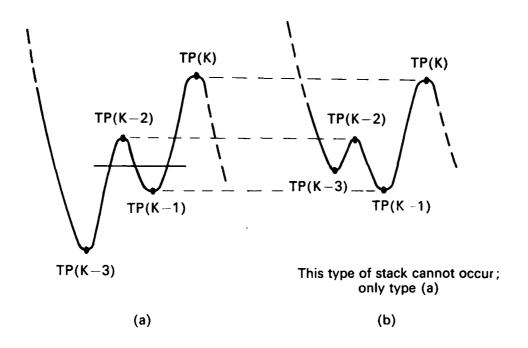


FIG. 3: DERIVATION OF THE THREE-POINT TEST— SEE CYCLE DEFINITION

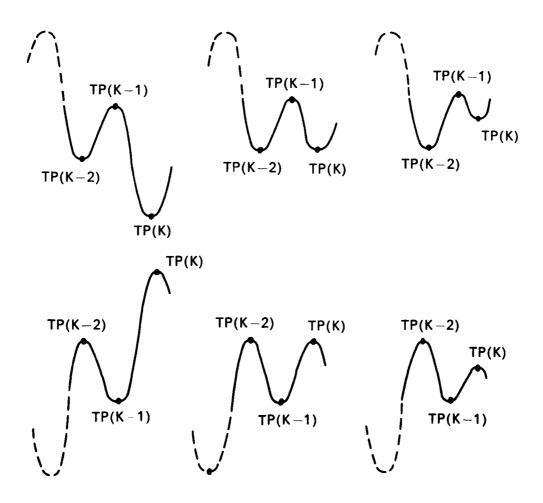
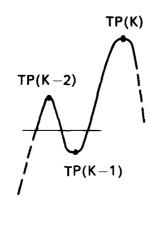
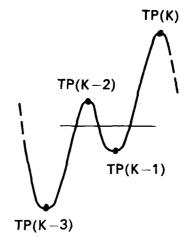
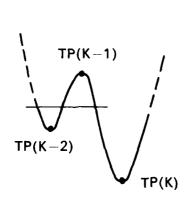
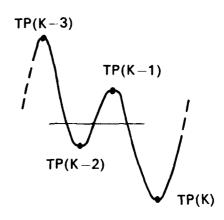


FIG. 4: THREE-POINT SEQUENCES







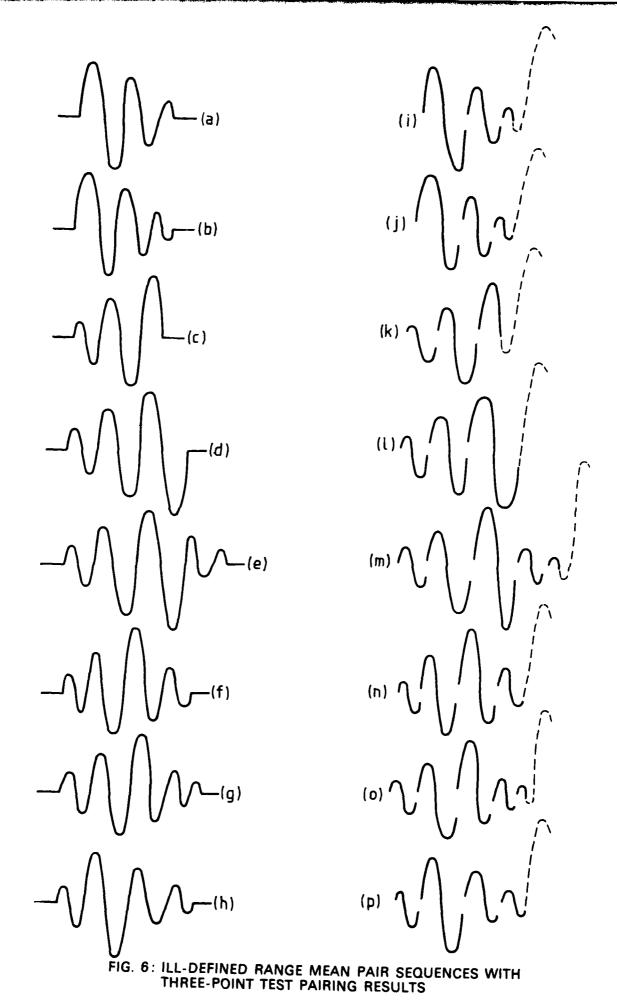


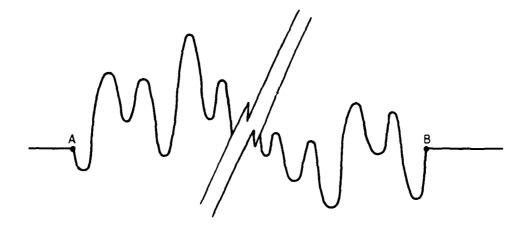
(a) The three-point test

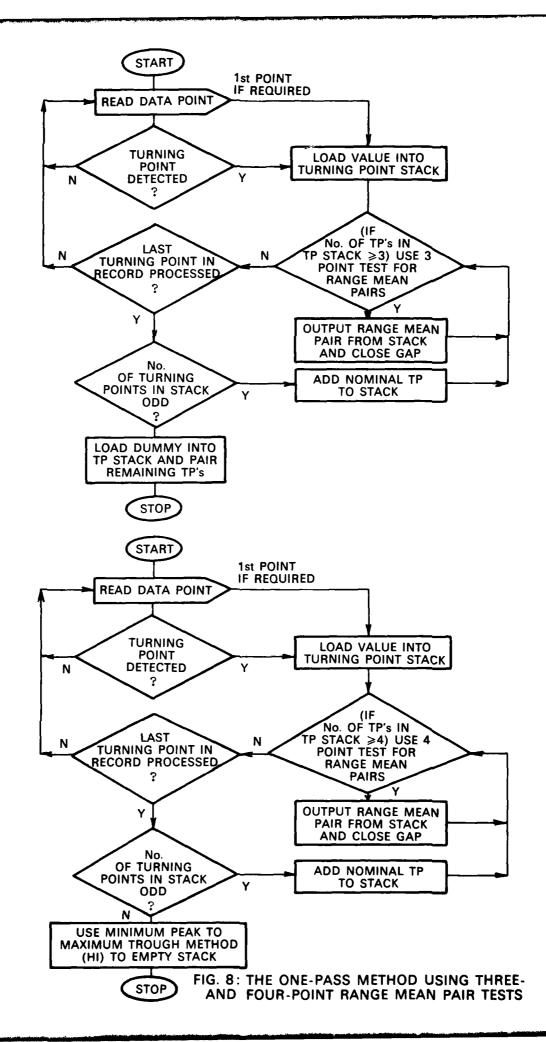
(b) The four-point test

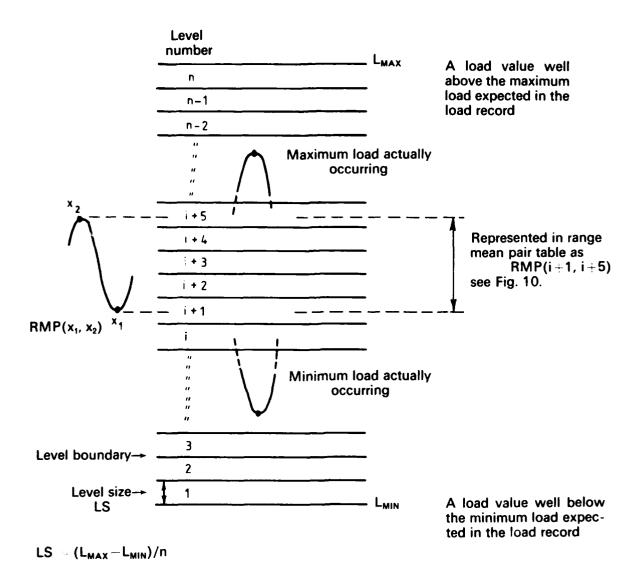
$$|\mathsf{TP}(\mathsf{K}-1)-\mathsf{TP}(\mathsf{K}-2)| \leqslant |\mathsf{TP}(\mathsf{K})-\mathsf{TP}(\mathsf{K}-1)|$$

$$|\operatorname{TP}(K-3) - \operatorname{TP}(K-2)| \geqslant |\operatorname{TP}(K-1) - \operatorname{TP}(K-2)| \leqslant |\operatorname{TP}(K) - \operatorname{TP}(K-1)|$$









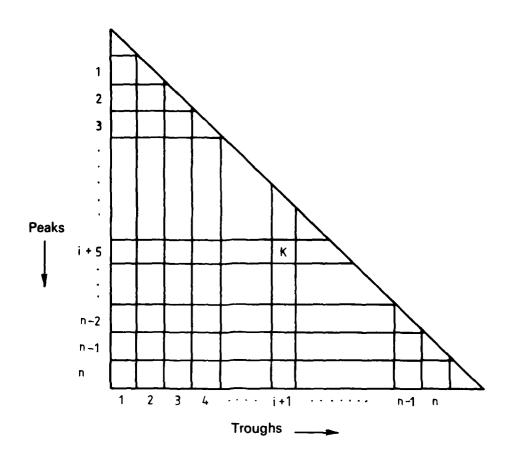


FIG. 10: THE RANGE MEAN PAIR TABLE

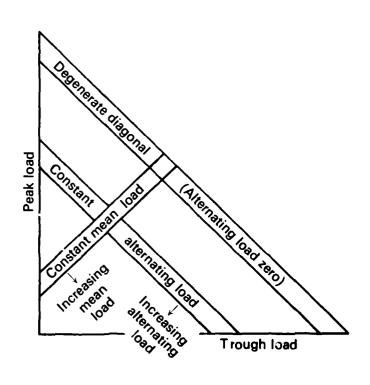


FIG. 11: RANGE MEAN PAIR TABLE CHARACTERISTICS

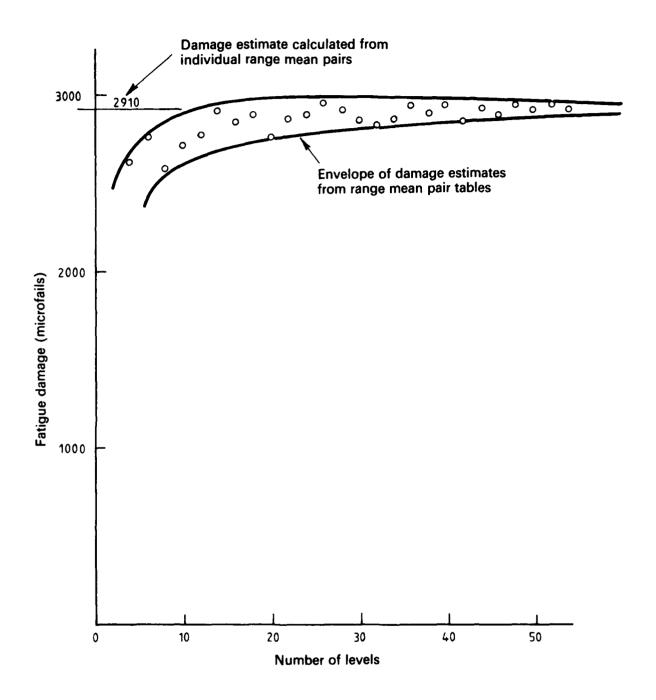
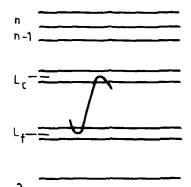


FIG. 12: EFFECT OF NUMBER OF LEVELS ON ACCURACY OF THE RANGE MEAN PAIR TABLE

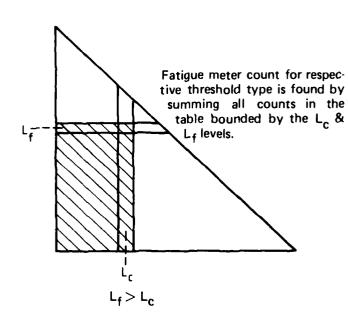
Fatigue meter threshold for which firing valve is greater than cocking valve i.e.  $L_{\rm f} > L_{\rm c}$ 

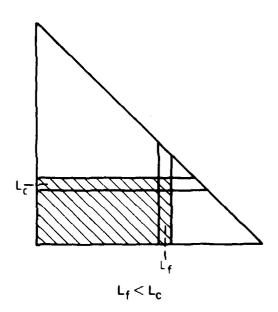
Fatigue meter threshold for which cocking valve is greater than firing valve i.e.  $\rm L_f < L_c$ 

n n-1	
L <sub>f</sub>	_
L <sub>c</sub> -	=
2	

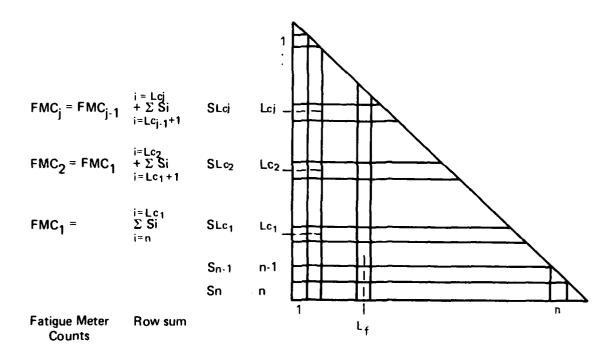


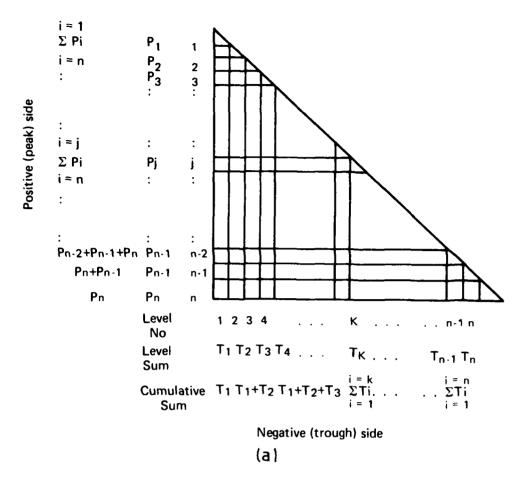
Minimum range mean pairs capable of registering a count for the given threshold type.





(b)





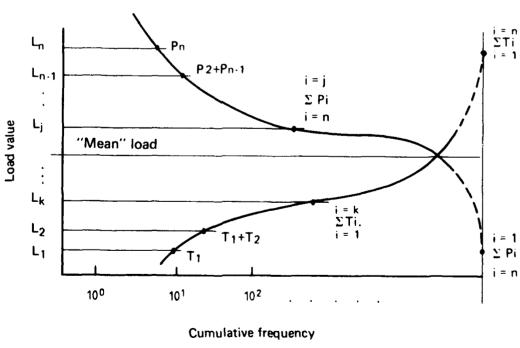


FIG. 15 - PRODUCING SPECTRA FROM RANGE MEAN PAIR TABLES.

(b)

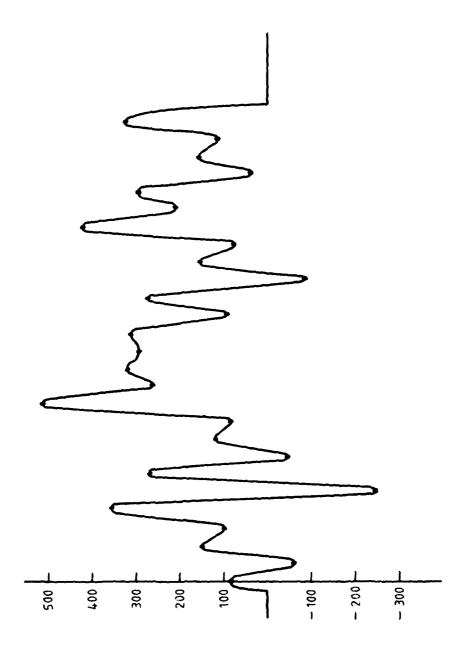


FIG. 16: TURNING POINT SEQUENCE FOR EXAMPLE 1

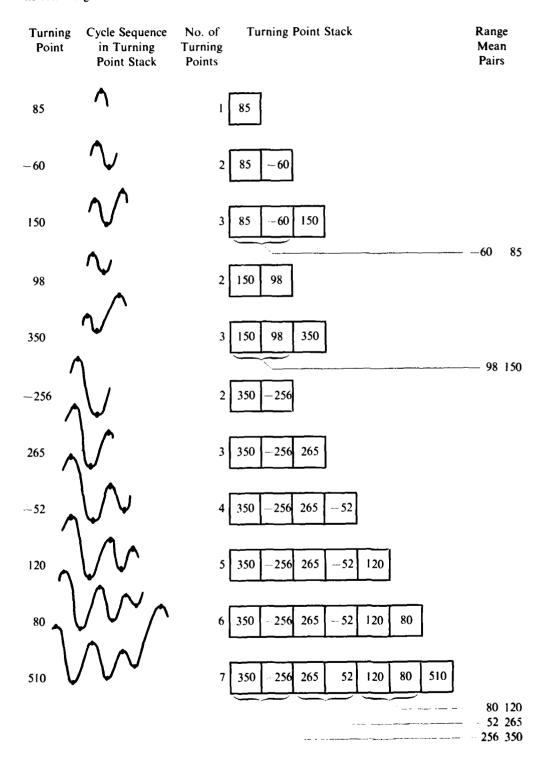
#### **APPENDIX**

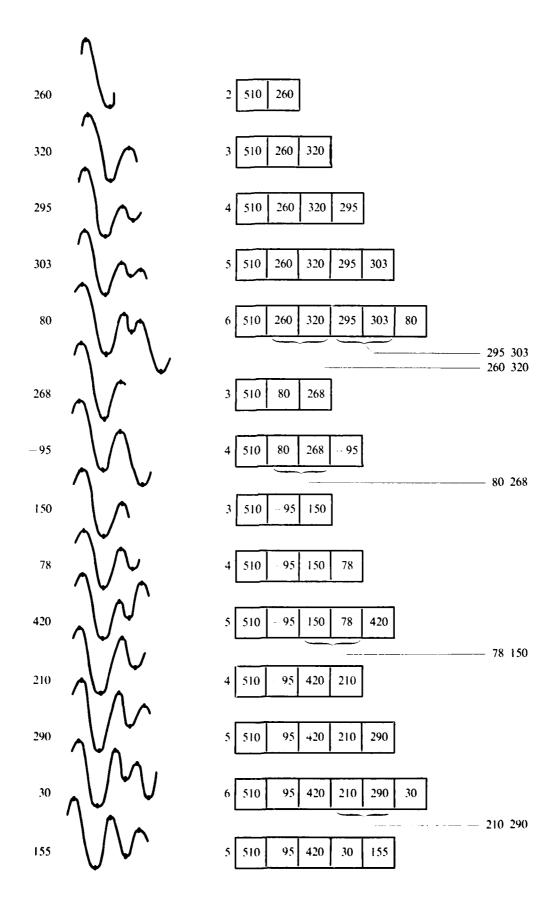
In the following pages two sample load histories are used to demonstrate the one pass range mean pair method. In the first sequence, shown in Figure 16, the procedure is outlined step by step using a schematic turning point vector. In the second example a more realistic counting situation is proposed.

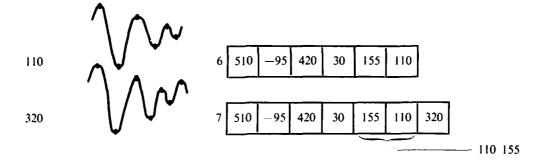
#### Example 1

The turning point sequence shown in Figure 16 is, in load terms:

Using the pairing procedure as given in section two for the three point one pass method the following is obtained:



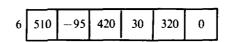




End Effect Correction:

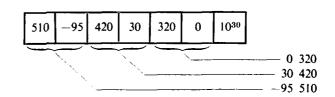
Odd number of TP's remaining — Add Nominal TP  $\phi = 0$ , say





Last TP is a trough ∴ Dummy TP = +1030





.. Range Mean Pairs obtained by Three Point Test:

Using the same procedure for a Four Point Test gives:

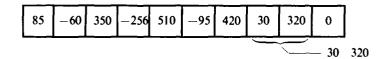
with the Turning Point Stack containing the following at the end of the sequence:



85	-60	350	<b>-256</b>	510	<b>-95</b>	420	30	320
----	-----	-----	-------------	-----	------------	-----	----	-----

End Effect Correction: Odd number of TP's remaining  $\therefore$  Add Nominal TP  $\phi=0$  say, and pair using peak-trough counting





0 85

85	-60	350	-256	510	-95	420	0

# Comparison of Results:

3 Point	t Test		4 Point	Test
98	150		98	150
80	120		80	120
-52	265		-52	265
295	303		295	303
260	320	=	260	320
80	268		80	268
78	150		78	150
210	290		210	290
110	155		110	155 J

Cycles which can be identified
as perturbations of larger cycles
i.e. range mean pairs.

-60		-60	350
-256	350	-256	350 510
0	320	0	85
30	420	30	320 420
<b>-95</b>	510	-95	420

Cycles which cannot be identified as perturbations of larger cycles.

## Example 2

The record shown overleaf contains two channels of data recorded during one flight of a monitored aircraft. The two channels, respectively normal acceleration by 100 and microstrain at an important location, have been processed using the three point one-pass method.

Records of both sets of range mean pair data are listed. However, as pointed out previously, range mean pair tables present a more condensed and convenient form for the same data. The table of range mean pair data for channel one is included.

•	LO+D FAC*34 100 m <sub>2</sub> (8)	STRAIN AT 98E (µ4)		LOAD FACTOR 100 Pt (6)	STAAIN AT 986 (µf)		LOAD FACTOR 190 N <sub>a</sub> (8)	\$TPAIN AT 986 (µ4)		LOAD STRAIN FACTOR AT 100 Ng 98E (8) (µ4)
3	130.83 135.24 137.69 135.24	9000.00 71.13 76.60 78.79 79.88	101 102 103 104 105	161.21 165.13 163.17 161.21 165.62	133.50 137.88 134.60 134.60 138.98	201 202 203 204 205	111.72 112.70 106.82 103.88 105.84	31.73 22.98 17.51 12.04 4.38	301 302 303 304 305	4.41 -247.31 2.94 -248.41 0.98 -248.41 4.41 -245.12 5.88 -240.75
7 8 9	141.12 140.63 139.65 145.04 143.08	80.98 83.17 87.54 90.83 94.11	106 107 108 109 110	162.68 163.66 168.07 164.15 165.13	137.88 138.98 138.98 140.07 141.16	206 207 208 209 210	99.47 99.47 98.98 91.63 92.61	4.38 0.00 -7.66 -17.51 -21.89	306 307 308 309 310	5.39 -240.75 8.82 -238.56 6.37 -237.46 10.29 -236.37 11.27 -235.27
14 15	145.04 149.45 145.04 150.92 152.39 147.49	95.20 99.58 102.86 106.15	111 112 113 114 115	168.07 161.70 165.62 166.60 161.70	136.79 135.69 137.88 137.88	211 212 213 214 215	89.67 85.26 86.73 77.91 74.48	-24.07 -28.45 -33.92 -45.96 -50.34 -56.90	311 312 313 314 315	9.31 -231.99 12.25 -229.80 14.21 -227.61 14.70 -222.14 16.66 -219.95
16 17 18 19 20 21	153.86 153.86 153.86 156.80 155.33	106.15 111.62 112.71 116.00 118.18 117.09	116 117 118 119 120 121	168.07 164.64 163.17 169.05 164.15 163.17	138.98 136.79 140.07 138.98 135.69	216 217 218 219 220 221	74.97 68.60 67.62 67.13 57.33 57.82	-58.00 -64.56 -73.32 -82.07 -88.64	316 317 318 319 320 321	16.66 -219.95 18.13 -215.58 23.52 -210.11 22.54 -205.73 24.01 -200.26 28.42 -195.88
22 23 24 25 26	156.31 158.27 155.82 159.25 159.25	118.18 117.09 119.28 120.37 122.56	122 123 124 125 126	165.62 162.68 162.68 163.17 158.76	133.50 130.22 134.60 135.69 133.50	222 223 224 225 226	54.88 49.49 49.00 43.12 38.22	-93.02 -95.20 -105.05 -118.18 -122.56	322 323 324 325 326	27.44 -195.88 30.38 -189.31 33.81 -180.56 34.79 -175.09 38.71 -170.71
27 28 29 30 31 32	156.80 161.70 158.76 159.25 162.68 159.25	122.56 123.66 123.66 125.84 131.32 128.03	127 128 129 130 131	164.15 162.19 157.29 162.68 158.76 157.78	132.41 130.22 128.03 128.03 123.66 126.94	227 228 229 230 231 232	34.30 32.83 33.32 26.95	-129.13 -132.41 -132.41 -140.07 -145.54 -148.82	327 328 329 330 331 332	43.12 -167.43 43.61 -157.58 49.49 -151.01 52.43 -143.35 55.86 -137.88 60.76 -131.32
33 34 35 36 37 38	161.21 162.19 157.29 160.72 162.68 160.72	126.94 122.56 120.37 126.94 132.41 131.32	133 134 135 136 137 138	159.25 154.35 157.78 157.29 153.37 157.78	126.94 121.47 121.47 119.28 122.56 121.47	233 234 235 236 237 238	18.13 20.09 15.19 12-25	-160.86 -166.33 -168.52 -173.99 -179.47 -184.94	333 334 335 336 337 338	63.21 -125.84 66.64 -116.00 71.05 -108.34 73.01 -98.49 79.38 -88.64
39 40 41 42 43	164.15 162.68 161.21 163.66 159.25	130.22 129.13 129.13 132.41 133.50	139 140 141 142 143	156.80 152.39 157.29 154.35 153.86	119.28 120.37 118.18 121.47 119.28	239 240 241 242 243	6. 37 4. 41 5. 88 0. 49 0. 98	-194.79 -196.97 -199.16 -204.63 -205.73	339 340 341 342 343	84.77 -82.07 88.20 -73.32 93.59 -63.47 95.06 -58.00 99.47 -47.05 105.35 -39.39
44 45 46 47 48	162.68 165.13 159.25 162.68 163.66 159.74	134.60 134.60 129.13 134.60 134.60	144 145 146 147 148 149	157.29 153.37 156.31 157.29 153.86 158.27	121.47 118.18 120.37 122.56 122.56 123.66	244 245 246 247 248 249	-4.90 -3.92 -5.39 -8.33	-212.29 -216.67 -219.95 -224.33 -227.61 -230.90	344 345 346 347 348 349	106.82 -31.73 111.23 -20.79 117.11 -14.23 119.56 -4.38 124.46 3.28 129.36 10.94
50 51 52 53 54	163.66 160.72 160.72 163.17 157.78	133.50 130.22 129.13 133.50 130.22	150 151 152 153 154	156.31 153.86 157.29 155.33 155.82	119.28 117.09 118.18 118.18 124.75	250 251 252 253 254	-11.27 -11.76 -12.25 -15.68 -13.72	-234.18 -242.93 -241.84 -247.31 -248.41	350 351 352 353 354	129.85 19.70 135.73 29.55 136.22 36.11 139.16 41.58 145.04 48.15
55 56 57 58 59 60	161.70 161.21 158.76 164.64 162.68 159.25	129.13 128.03 130.22 132.41 137.68 132.41	155 156 157 158 159 160	160.72 157.78 159.74 162.68 160.23 163.66	126.94 123.66 128.03 139.22 133.50 130.22	253 256 257 258 259 260	-19.11 -18.13 -19.11 -20.09	-249.50 -254.97 -259.35 -261.54 -264.82 -265.91	355 356 357 358 359 360	144.55 52.53 148.47 63.47 153.86 70.04 152.88 75.51 157.29 84.26 160.23 90.83
61 62 63 64 65	165.13 161.21 162.19 166.11 163.17 164.15	131.32 132.41 136.79 138.98 136.79 133.50	161 162 163 164 165	162.19 159.74 162.68 158.76 160.72 164.15	131.32 128.03 129.13 130.22 130.22	261 262 263 264 265	-22.54 -21.07 -23.03 -22.54	-267.01 -270.29 -271.39 -275.76 -279.05	361 362 363 364 365	160.72 97.39 164.64 106.15 165.62 109.43 166.60 118.18 171.01 126.94 169.54 131.32
66 67 68 69 70 71	165.13 161.21 166.60 165.62 162.19	135.69 138.98 140.07 137.88 137.88	166 167 168 169 170 171	159.25 160.23 162.19 156.80 159.25	130.22 130.22 126.94 125.84 1293	266 267 268 269 270 271	-23.03 -20.58 -23.52 -22.54 -21.07	-272.48 -275.76 -275.76 -277.95 -277.95 -279.05	366 367 368 369 370 371	173.46 135.69 176.40 138.98 174.93 146.64 178.36 149.92 178.95 151.01
72 73 74 75 76	164.15 162.19 163.66 165.62 159.74 164.15	135.69 135.69 136.79 137.88 136.79 135.69	172 173 174 175 176 177	159.25 155.82 159.25 157.78 155.33 157.29	126.94 125.84 128.03 126.94 129.13 128.03	272 273 274 275 276 277	-19.60 -18.13 -20.58 -17.64	-276.86 -277.95 -275.76 -275.76 -279.95 -276.86	372 373 374 375 376 377	175.91 148.82 180.81 152.11 178.36 153.20 179.83 157.58 181.30 159.77 177.38 157.58
78 79 80 81 82	1615 159.25 164.15 162.68 161.21	132.41 130.22 132.41 133.50 140.07	178 179 180 181 182	152.39 152.8R 155.33 147.49 147.98	125.84 125.84 123.66 121.47 116.00	278 279 280 281 282	-17.64 -13.72 -14.70 -14.70 -12.74	-275.76 -274.67 -275.76 -272.48 -275.76	378 379 380 381 382	178.36 157.58 178.36 156.48 175.42 157.58 178.85 160.86 177.87 166.33
83 84 85 86 87 88	167.58 163.66 164.64 167.09 163.17 167.09	136.79 140.07 141.16 141.16	183 184 185 186 187 188	148.47 140.14 139.16 139.18 134.75 135.71	111.62 102.86 99.58 95.20 85.64 84.26	283 284 285 286 287 288	-13.72 -9.80 -11.27 -9.80 -7.35	-274.67 -272.48 -272.48 -269.20 -271.39 -269.20	3A3 3R4 3B5 3R6 387 3BR	176.89 1615 181.79 175.71 178.36 175.09 180.32 179.47 182.77 179.47 179.34 179.47
89 90 91 92 93	167.09 163.66 168.56 166.11 161.70	138.98 140.07 137.88 136.79 136.79	189 190 191 192 193	129.85 128.87 133.28 126.42 127.42 126.42	78.79 74.41 72.22 64.56 60 19 58.00	289 290 291 292 293 294	-10.29 -7.84 -6.37 -6.85 -4.41	-268.10 -268.10 -263.73 -260.44 -260.44 -259.35	389 390 391 392 393 394	182.77 180.56 183.26 182.75 180.32 183.84 182.28 178.37 180.81 176.18 177.87 173.99
95 96 97 98 99 100	163.66 164.15 167.09 161.70 162.68	138.98 138.98 136.79 131.32 134.60	195 196 197 198 199 200	121.03 123.48 121.03 117.11 119.56 116.17	52.53 50.34 45.96 42.68 36.11 31.73	295 296 297 298 299 300	-5. 39 -1. 95 -2. 45 1. 47 1. 95	-261.54 -257.16 -251.69 -251.69 -748.41 -249.50	395 396 397 398 399 400	181.79 179.37 177.87 181.65 179.34 178.37 180.32 176.18 177.38 182.75 183.75 180.03

X X X

401	1 184.73 191									
40			1 155.33	119.13	621 622	108.75	31.73	731	105, 35	19.55
40	3 192.57 199	51	3 156. 40	118.18	623		32.83 31.73	732 733	103.88	20.79 17.51
404		51		118.18	624	110.25	31.73	734	104.37	18.60
405 406		n3 5t		119.29	62.5	113.48	31. 3	735	105.84	21.89
40			7 152.39	117.19	62 6 62 7	112.70	32.83	736	107.80	19.70
408	202.37 219	1. 56 51		119.19	62.5	114.17	33. /2 33. 92	737 718	100.94	14.23 12.04
409		- 14 51		113.81	629	111.72	30.64	739	103.88	14.23
410				121.47	630	IC8.78	26.26	740	102 - 90	14.23
412		18 52 1-56 52		122.56	631 632	113.55	29. 55 33. 92	741	107.31	16.41
413	712.66 239			116.00	633	113.58	40.49	74.2 74.3	104.86	15.32 12.04
414		. 84 52.		1!3.3!	634	117.11	38. 30	744	104.37	12.04
415		. 94 52  - 41 52		109.43	635	112.21	31.73	745	100.94	10.94
417		1.41 520 1.79 52		112.7L 112.7L	636 637	113.4d 113.6d	28.45 30.64	746	101.92	10.94
41	220-50 253	. 88 521		110.52	633	112.21	33.92	74 <i>7</i> 748	105.35	9.85 10.94
419		.97 529		104.34	639	115.52	38. 30	749	105.35	12.04
421		. 07 530 - 35 53		: 15-05 106-15	640	113.19	33.92	350	104.37	16.41
422	227-36 260			107.24	6-1 642	112.21	29.55 27.36	751 752	100.94	13.13 9.45
423		. 25 53:	3 149.45	101.77	64.3	111.23	31.73	753	101.92	6.57
424				98.49	644	112.10	36.11	754	101.42	10. 4
426		. 63 535 . 73 536		102.86 99.58	645 646	116.62	35.02 29.55	755	105.35	13.13
427	226.87 265			100.68	647	113.58	28.45	756 757	102.90	15. 32 13.13
428		.73 538		96.30	645	114.17	31.73	753	104.86	8. 75
429 430		-73 539 -73 540		93 - 02 95 - 20	649	109.76	31.73	759	101.92	8. 75
431	228.34 264	. 82 541		96-30	650 651	113.68	30.64 29.55	760 761	104.37 103.98	12.04 13.13
432		.OL 542	145.04	98. 49	652	110.25	28.45	762	100.45	10.94
433 434				101 - 77	653	114.17	29.55	763	103.86	7. 56
435	222.46 264			99.58 102.86	654 655	110.74	29.55 27.36	764 765	101.43	9.85
4 36	225-89 264	. 82 54 6	150.92	107.24	656	113.68	30.64	765 766	100.94	7. 66 4. 38
437				105-15	657	109.76	28.45	767	97.51	3- 28
439				102.86 100.68	658 659	112.21	29.55	768	98. 99	3.28
440	220.99 263			94-11	660	108.29	30. 64 27. 36	769 770	95.55	4.38 3.28
441	223-93 250	. 14 551	147.00	98. 49	661	113.19	28.45	771	100.45	4. 38
442				100.68	662	111.23	28.45	712	98.00	1.09
444				93.02 89.73	663 664	110.25	29.55	773	95.06	0.00
445	220.01 258			89.73	665	114.17	30.64 29.55	774 775	99. 47 95. 55	3. 28 0. 00
446				90.83	666	109.27	26. 26	776	95.06	1.09
447				89.73	667	113.19	28.45	777	989	-1.09
449				78. 79 74. 41	668 669	108.78	29.55 28.45	778 779	94.08	-5. 47
450	219-03 253	.98 560	133.77	75. 51	670	113.68	30.64	780	95.06 95.06	-3.47 -5.47
451 452	215-11 251			76. 10	671	110-25	29.55	781	91.63	-5.47
453	215.09 252. 216.58 251.		134.75 132.30	76.50 71.13	672	112.21	285	782	94.08	-7.66
454	215.60 251			68.94	673 674	111.23	27.36 30.64	783 784	93.59	-7.55
455	217-07 248.	. 41 565	130.34	68.94	675	115.15	32.83	785	90.16 95.06	-6.57 -7.66
456 457	214.13 245.			68. 94	676	113.19	31.73	786	90.65	-9.85
458	214.13 246. 218.05 242.			66.75 59.09	677 678	115.15	36.11	787	90-16	-8. 75
459	214-13 240.			60.19	679	117.11	38.30 41.58	786 789	94 - 08 87 - 71	-9.85 -8.75
460	214.13 239.			59. 09	680	118.58	40.49	790	88. 20	-10.94
461 462	217.07 238. 213.15 235.		124.95 122.01	61.47	681	118.58	37.21	79 L	91.63	-14.23
463	215.60 235.		125.93	68. 94 63. 47	682 683	113.68	40. 49 41. 58	792 793	87.22 90.65	-12.04 -13.13
464	214.13 234.		122.01	54.72	684	116.13	40.49	794	91 . 14	-12.04
465 466	213.64 233.		119.56	52.53	685	114-17	39. 39	795	88. 20	-12.04
467	215.11 231. 214.13 235.		123.48 121.03	56.90 62.38	686 687	117.11	29.55 24.07	796	92 - 61	-12.04
468	215.11 234.	18 578	124.46	66. 75	688	109.76	22.98	79 7 79 8	90.16 87.71	-10.94 -12.04
469	220.50 235.			58.00	689	112.70	20. 79	799	93.10	-14.23
471	216.58 240. 219.03 240.		120.05 122.01	52.53		108.78	25.17	800		-13.13
472	220.99 240.	75 582		50. 34 53. 62	691 692	109.75 110.25	28. 45 22. 98	801 802	92 - 12 95 - 06	-9.85 -10.94
473	218.54 238.	56 583	119.07	54.72	693	105.35	17.51	803	89.67	-8.75
474 475	220.50 240. 219.52 240.		122.01	54. 72	694	106.33	12.04	804	92 . 61	-6.57
476	217.56 239.		117.11 115.15	51.43 45.96	695 696	104.37	10.94 15.32	805 806	93. 59 89. 67	-9. 85 -9. 85
477	220.01 240.	75 587	117.11	41.58	697	107.31	17.51	807	96.53	-9.83 -6.57
478	217.07 240.	75 588	112.21	42.68	698	103.88	18.60	808	93.59	-8.75
480	216.09 239. 214.62 236.		114.17 116.62	45.96 44.87	699 700	102.90 106.82	16.41 12.04	809	92 · 12 96 · 53	-7.66
481	209.23 235.	27 591	110.25	42.68	701	99.96	12.04	81 0 81 1	96.33	-5, 47 -6, 57
482	209.72 229.	80 592	112.21	39. 39	702	102.90	17.51	B1 2	94.08	-2 . 19
483 484	209.72 227. 205.31 226.	61 593 52 594		33.92	703	106.82	16.41	81.3	98.00	-3.28
	204.33 222.	14 595		38. 30 39. 39	704 705	100.45	14.23 16.41	81 4 81 5	93.10 96.04	→. 38 -4. 38
486	200.90 216.	67 596	110.25	39. 39	706	103.59	13.13	81.0	98.00	-5, 47
487 488	197.47 214.	48 597	110.25	36.11	707	102.93	16.41	61.7	93.59	-3.28
489	198.45 209.		115.64 111.72	33.92 30.64	708 709	106.33	25.17 22.98	81 8 81 9	98. 49 96. 04	-3.78
490	191.59 204.	63 600	111.72	36.11		103.88	19.70	820	93.59	-3.28 -2.19
491 492	190.61 200.			38. 30	71.1	105.35	6. 57	821	99.96	-1.09
493	183.75 195. 184.24 191.			35.02 35.02	71 2 71 3	98.00 99.76	-2.19 1.09	822	95.06	-3.28
494	181.79 179.	47 604	113.19	33.92		101 - 43	8.75	82 3 82 4	96.04 99.47	-1 · 09 0 · 00
495	176.40 178.	37 605	110.74	39. 39	71 5	98.98	13.13	02 5	94.57	-4.38
496 497	177.87 176. 176.40 171.		116.13	43.77		105.35	19.70	<b>8</b> 2 6	97.02	-1 - 09
498	171.99 166.		113.15 114.17	40.49 40.49		106.33 104.37	16. 41 17. 51	82 7 62 8	97.51 94.57	-3.28
499	171.50 159.	77 609	118.58	36.11		109.76	20. 79	82 9	99.47	-2.19 1.09
	167-09 155.	39 610	112.21	39. 39	720	103.84	19.70	#10	97.02	-3.28
501 502	166.11 153.1		115.15 117.60	41.58 41.58	721	104.86	18.60	831	95.06	-3.28
503	160.72 143.		111.72	41.58 36.11		107.80 104.37	17.51 16.41	832 813	97. 31 95. 06	-4.38 -2.19
504	160.72 138.9	98 614	116.62	36.11	72 4	105.84	19. 70	834	95.55	-3.70
505 506	161.70 135.6 156.80 131.3		115.64	33. 92	725	109.76	25.17	835	97. 51	-3.28
507	158.76 130.		110.25 115.64	35.02 36.11		103.88	20. 79 18. 60	834 837	94.57 93.55	-3.28 -4.30
508	155. 82 123.6	61 8	113.19	35.02		107.31	16.60	837 838	97.07	-5. 47
309 310	154.35 125.6 159.25 124.3		111.72	31.73	729	104.37	18.60	639	93.10	-4.38
		620	115.64	32.8)	730	109.27	20. 79	840	97. 31	<b>-4.38</b>

A 45

⋖
-
⋖
۵
~
=
⋖
4
7
7
$\overline{\omega}$
₹
Σ
щ
9
Z
⋖
Œ

0006	
*DUMYY	
SEQUENCE BY	
NOTE CLOSED	
GAUGE 9BE N 120. 3730 129. 1274 128. 0331 131. 3160 133. 5046 133. 5046 133. 5046 131. 3160 135. 6932 123. 6559 133. 5046 141. 1647 240. 7460 122. 5616 100. 6756 107. 2414 68. 9409 66. 7523 39. 3948 41. 5834 43. 7720 38. 3005 36. 1119 40. 4891 41. 5834 28. 4518 12. 0373 25. 1689 16. 4145	15.3202 25.1689 1.0943 267.0092
CHANNEL 6: 131.5989 134.5989 134.5989 137.8818 138.9761 140.0704 141.1647 130.2217 118.1844 117.0901 71.1295 -279.0465 181.6538 183.8424 231.9916 113.8072 94.1098 93.0155 59.0922 52.5264 33.9233 36.1119 30.6404 28.4518 27.3575 26.2632 26.2632 26.2632 26.2632 26.2632	6.5658 -2.1886 -14.2259 -279.0465
x100	
e	
T CHANNEL 1	
NGE-PAIR COUNT 157. 2900 157. 2900 157. 2900 167. 0900 161. 0900 161. 7000 161. 2100 164. 1500 164. 1500 164. 1500 164. 1500 164. 1500 175. 4200 177. 3800 228. 8300 228. 8300 228. 8300 177. 3800 117. 1100 118. 5800 117. 1100 118. 5800 119. 0700 119. 0700 106. 3300 106. 8200 106. 8200	107.3100 109.7600 89.6700 99.9600 229.3200
DATA FOR RANGE—PAIR 162.6800 157.2900 164.6400 157.2900 165.1300 157.2900 166.6000 159.2500 161.7000 167.0900 168.5600 161.2100 168.5600 161.2100 158.7600 164.1500 157.2900 162.6800 157.2900 162.6800 157.2900 162.6800 157.2900 162.6800 157.2900 162.0900 157.2900 162.6800 157.2900 162.500 181.3000 177.3800 223.4400 228.8300 223.4400 228.8300 110.2500 117.6000 110.2500 115.6400 110.2500 115.6400 110.2500 111.100 110.2500 114.1700 110.2500 114.1700 110.2500 114.1700 110.2500 114.1700 110.2500 114.1700 110.2500 114.1700 110.2500 114.1700 110.2500 116.6200 110.4300 107.3100 100.4500 106.8200 110.7000 109.2700	100.9400 98.0000 95.0600 87.2200 -23.5200

246.																								0	0
234. 2																							0	0	0
222. 2																						-	0	0	0
210. 2																					1	0	0	0	0
																				0	0	0	0	0	0
186. 1																			0	0	0	0	0	0	0
74. 1																		2	0	0	0	0	0	0	0
62. 1																	0	0	0	0	0	0	0	0	0
150. 1																-	10	0	0	0	0	0	0	0	0
138.															0		0	ပ	0	0	0	0	0	0	0
126.														0	0	0	_	0	0	0	0	0	0	0	0
114. 126. 138. 150. 162. 174. 186. 198.													-	0	0	0	0	0	0	0	0	0	0	0	0
102.												7	<b>∞</b>	0	0	0	0	0	0	0	0	0	0	0	0
90.											0	S	0	0	0	0	0	0	0	0	0	0	0	0	0
78.										0	7	0	0	0	0	0	0	0	0	0	0	0	0	0	0
.99									0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
54.								0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
42.							0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
30.						0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
18.					0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
•				0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
9			0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
-18.		0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
-3018.	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	-	0	0	0
								54.																	

TOTAL NUMBER OF RANGE PAIRS = 36

RANGE PAIR TABLE FOR DATA FROM CHANNEL 1. ie., Nz  $\times$  100.

## **DISTRIBUTION**

#### **AUSTRALIA**

Department of Defence	Copy No.
Central Office	
Chief Defence Scientist	1
Deputy Chief Defence Scientist	2
Superintendent, Science and Technology Programs	3
Australian Defence Scientific and Technical Representative (UK)	4
Counsellor, Defence Science (USA) Defence Library	5
Joint Intelligence Organisation	7
Assistant Secretary, D.I.S.B.	8–23
Aeronautical Research Laboratories	
Chief Superintendent	24
Superintendent, Structures Division	25
Structures Divisional File	26
Author: R. C. Fraser	27
Library D. G. Ford	28 29
P. J. Howard	30
D. J. Sherman	31
D. Graham	32
Materials Research Laboratories	
Library	33
Defence Research Centre, Salisbury	•
Library	34
Central Studies Establishment Information Centre	35
	33
Engineering Development Establishment Library	36
·	50
RAN Research Laboratory Library	37
	3,
Navy Office Naval Scientific Adviser	38
Army Scientific Adviser	
Army Scientific Adviser	39
Royal Military College	40
US Army Standardisation Group	41
Air Force Office	
Air Force Scientific Adviser	42
Aircraft Research and Development Unit	43
Technical Division Library D. Air Eng	44 45
HQ Support Command (SENGSO)	46

Department of Productivity Government Aircraft Fac		
Library		47
R. Hutton		48
M. Cameron		49
Department of Transport		
Secretary		50
Library		51
Airworthiness Group	: C. Torkington	52
	R. Douglas	53
Statutory, State Authortitie	es and Industry	
-	nergy Commission, Director	54
	ngineering Division, Chief	55
CSIRO Materials Scientification	= -	56
Qantas, Library		57
Trans Australia Airlir	nes, Library	58
Ansett Airlines of Au		59
BHP Central Research	h Laboratories, N.S.W.	60
BHP Melbourne Rese	arch Laboratories	61
Commonwealth Aircr	aft Corporation:	
H. Granow		62
R. Keays		63
R. Beckett		64
Hawker de Havilland	Pty. Ltd., Librarian, Bankstown	65
Universities and Colleges	•	
Adelaide	Professor of Mechanical Engineering	66
Australian National	Library	67
Flinders	Library	68
James Cook	Library	69
Latrobe	Library	70
Melbourne	Professor Whitten, Mechanical Engineering	71
Monash	Library	72
	Professor Polmear, Materials Engineering	73
Newcastle	Library	74
New England	Library	75
New South Wales	Professor R. A. A. Bryant, Mech. and Ind. Eng.	76
	Assoc. Professor R. W. Traill-Nash, Struc. Eng.	77
Queensland	Library	78
Sydney	Professor G. A. Bird, Aeronautical Engineering	79
	The Head, Mechanical Engineering	80
Tasmania	Engineering Library	81
West Australia	Library	82
R.M.1.T.	Library	83
	Mr H. Millicer, Aeronautical Engineering	84
CANADA		
CAARC Co-ordinator	r Structures	85
	nautics Establishment, Library	86
Universities		
McGill	Library	87
Toronto	Institute for Aerospace Studies	88
Waterloo	Department of Mechanical Engineering	89
	Department of Civil Engineering	90

FRA	NCE			
	AGARD	Library	91	
	AMD-BA	D. Chaumette	92	
		M. Peyrony	93	
	ONEKA	Library	94	
	Service de Documentation, Technique de l'Aeronautique			
GER	MANY			
	Deutsche Luft-und	Raumfahrt	96	
	ZLDI		97	
	Industrie-anlagen E		98	
	Laboratorium fur l	Betriebsfestigkeit	99	
INDI				
	CAARC Co-ordina		100	
	Civil Aviation Dep		101	
		Aero Development Establishment, Library	102	
	Hindustan Aeronai		103	
		Technology, Library	104	
	National Aeronaut	ical Laboratory, Director	105	
INTE	ERNATIONAL CON	MMITTEE ON AERONAUTICAL FATIGUE		
	Through Australia	n Representative	106–130	
ISRA	<b>NEL</b>			
	Technion-Israel I	nstitute of Technology, Professor J. Singer	131	
ITAL	LY.			
	Associazione Italia	na di Aeronautica e Astronautica, Professor Evla	132	
	Aeronautica Macc	hi, Dr Bazocchi	133	
1	Universities			
	Pisa	Dr Salvetti	134	
JAP.	AN			
	National Aerospac	e Laboratory, Library	135	
1	Universities			
	Tohoku (Sendai)	Library	136	
	Tokyo	Institute of Space and Aeroscience	137	
	<b>3</b> ·	•		
NET	HERLANDS		120	
		pplied Science Research TNO, Library	138	
	National Aerospac	e Laboratory (NLR) Library	139	
NEW	V ZEALAND			
	Defence Scientific	Establishment, Library	140	
	Transport Ministry	y, Civil Aviation Division, Library	141	
	Universities			
	Canterbury	Library	142	
		Mr F. Fahy, Mechanical Engineering	143	
SWI	EDEN			
	Aeronautical Rese	arch Institute	144	
	Chalmers Institute of Technology, Library			
Kungliga Tekniska Hogskolan				
	SAAB-Scania, Lib	гагу	147	

SWITZERLAND		
Swiss Federal Aircra	ft Factory, F + W;	
A. Jordi		148
R. Kaelin		149
UNITED KINGDOM		
Aeronautical Research	ch Council, Secretary	150
CAARC, Secretary		151
	lishment, Library, Farnborough	152
Military Vehicles Engineering Establishment		153
National Engineering Laboratories, Superintendent		154
British Library, Science Reference Library		155
British Library, Lending Division		156
	Research Establishment, Superintendent	157
CAARC Co-ordinate		158
Aircraft Research As	· · · · · · · · · · · · · · · · · · ·	159
	Metals Research Association	160
British Ship Research		161
	titute Ltd, Research Director	162
	search Association, Director	163
Science Museum Lib		164
Welding Institute, Li	•	165
	orporation, Military Aircraft Division	166
	orporation, Commercial Aviation Division	167
Short Brothers Ltd	orporation Ltd, Library	168 169
Westland Helicopters	- 1 14	170
westiand Tiencopiers	s Liu	170
Universities and College		
Bristol	Library, Engineering Department	171
Cambridge	Library, Engineering Department	172
London	Professor A. D. Young, Aero. Engineering	173
Manchester Library		174
Nottingham	Library	175
Sheffield	Library, Engineering Department	176
Southampton	Library	177
Strathclyde	Library	178
Cranfield Institute	L'II.	170
of Technology	Library	179
Imperial College	The Head	180
	Department of Aeronautical Engineering	181
UNITED STATES OF AN	MERICA	
	Technical Information Facility	182
Sandia Group Resea		183
	f Aeronautics and Astronautics	184
Applied Mechanics I		185
The John Crerar Lib	•	186
Boeing Co. Head Of		187
Cessna Aircraft Co.,		188
Lockheed Aircraft C		189
	Corporation, Director	190
Battelle Memorial In	•	191
Calspan Corporation		192
Universities		
Arizona	Professor B. J. Johnstone, Civil Engineering	193
California	Dr M. Holt, Department of Aerosciences	194
Florida	Aero. Engineering Department	195

The State of the S

Harvard	Professor G. F. Carrier, Applied Mathematics	196
Johns Hopkins	Professor S. Corrsin, Mechnical Engineering	197
Princeton	Professor G. L. Mellor, Mechanics	198
Stanford	Department of Aeronautics, Library	199
Polytechnic Institute of New York	Aeronautical Laboratories Library	200
California Institute	•	
of Technology	Guggenheim Aeronautical Labs, Library	201
Massachusetts Inst.		
of Technology	Library	202
Spares		203_220